

. 16

# RECOVERY FORCE ADJUSTMENT DEVICE

## FOR A PADDLE OF A CYMBAL STAND

## **BACKGROUND OF THE INVENTION**

#### 1. Field of the Invention

The present invention relates to a recovery force adjustment device, and more particularly to a recovery force adjustment device for a paddle of a cymbal stand so that the user is able to easily adjust the magnitude of the paddle recovery force to adapt to different performers with different habits

## 2. Description of Related Art

With reference to Fig. 7, a conventional adjusting device for changing the magnitude of the recovery force from the transmission rod (50) to the paddle (60) of a cymbal stand (70) includes a bonding block (53) securely connecting the first rod (51) to the second rod (52) of the transmission rod (50), a stop (54) formed on a mediate portion of the first rod (51) and having a disk (56) extending from the stop (54) and a recovery spring (55) mounted around the first rod (51) and sandwiched between the disk (56) and the bonding block (53). A sleeve (57) is screwingly connected to the outer periphery of the cymbal stand (70) and having an indentation (not numbered) defined to receive therein the disk (56).

When the recovery force of the recovery spring (55) is required to change, the user is able to rotate the sleeve (57) to change the distance between the disk (56) and the bonding block (53). When the distance between the disk (56) and the bonding block (53) is changed, the extension length of the recovery spring (55) is also changed. Therefore, the force required to move the paddle (60) downward relative to the cymbal stand (70) is changed.

- This adjustment of the recovery force requires the user to continuously
- 2 rotate the sleeve (57) until the required recovery force magnitude is reached.
- 3 That is, the adjustment process of the magnitude of the recovery force is
- 4 troublesome and inefficient.

- 5 To overcome the shortcomings, the present invention tends to provide an
- 6 improved recovery- force adjusting device for a paddle of a cymbal stand to
- 7 mitigate the aforementioned problems.

# SUMMARY OF THE INVENTION . . .

The primary objective of the present invention is to provide an improved recovery force- adjusting device to allow the user to easily adjust the magnitude of the recovery force from the recovery spring to the paddle of the cymbal stand.

Another objective of the present invention is to provide a handle pivotally mounted on the supporting seat of the cymbal stand, a driving ring movably mounted around a column of the cymbal stand and extending through the column and a rotating sleeve rotatably mounted around the column. The driving ring has two extensions oppositely extending toward the rotating sleeve which has a set of first teeth formed on a bottom portion of an outer periphery of the rotating sleeve, a set of second teeth and a set of third teeth formed on the outer periphery of the rotating sleeve, wherein both the second set of teeth and the third set of teeth are inclined with respect to the first set of teeth and oppositely correspond to each other. The supporting seat has a wedge formed on an inner face of the supporting seat to correspond to a tooth of the first set of teeth of the rotating sleeve. Therefore, when the two extensions are positioned in teeth of both the second and third sets of teeth, the wedge is on a slope of a tooth

- of the first set of teeth and when the handle is pivoted to force the wedge to be
- 2 positioned in the tooth of the first set of teeth, the sliding movement of the wedge
- 3 along the slope of the tooth of the first set of teeth forces the rotating sleeve to
- 4 rotate, which allows the two extensions to correspond to and be received in
- 5 different two teeth respectively of the second set of teeth and the third set of teeth.
- 6 Thus the recovery force from the recovery spring is changed.
- 7 Other objects, advantages and novel features of the invention will
- 8 become more apparent from the following detailed description when taken in
- 9 conjunction with the accompanying drawings.

# 10 BRIEF DESCRIPTION OF THE DRAWINGS

Fig. 1 is a perspective view of the cymbal stand with the recovery force

- 12 adjusting device of the present invention;
- Fig. 2 is a side plan view showing the recovery force adjusting device of
- 14 the present invention;
- Figs. 3 and 4 are schematic side plan views in partial section, wherein
- when the wedge is on a slope of a tooth of the first set of teeth, the two extensions
- are located in teeth of the second and the third sets of teeth;
- Figs. 5 and 6 are schematic side plan views in partial section showing
- 19 that when the handle is pivoted to drive the driving ring upward relative to the
- supporting ring, the wedge is received in a corresponding tooth of the first set of
- 21 teeth; and
- Fig. 7 is a schematic cross sectional view of a conventional recovery
- force adjusting device for a cymbal stand.
- 24 DETAILED DESCRIPTION OF THE PREFERRED EMBODIMENT

With reference to Fig. 1, a cymbal stand (10) has a paddle (20) movably 1 2 connected to the cymbal stand (10) via a transmission rod (11) in a column (not 3 numbered) such that a cymbal (not shown) on top of the transmission rod (11) is 4 driven by the movement of the paddle (20). It is to be appreciated that the mechanism for the transmission rod (11) to resume its original position after 5 6 movement of the paddle (20) is the spring (as shown in Fig. 3) which is mounted 7 around the transmission rod (11). The spring has a first distal end securely 8 connected to the paddle (20) and a second distal end securely connected to a stop 9 movably received in the column and mounted outside the transmission rod (11). 10 Because the recovery mechanism of the cymbal stand (10) is conventional in the art, detailed description of the interaction of the paddle (20) and the recovery 11 12 spring, as well as the stop is omitted for brevity. With reference to Figs. 2 and 3, the recovery force adjusting device in 13 14 accordance with the present invention has a supporting seat (40) securely 15 mounted outside the column (100) of the cymbal stand (10) and has a cutout (400) 16 defined in the supporting seat (40) to receive therein a driving ring (30). The driving ring (30) has a pin (14), and two extensions (31) oppositely extending 18 downward. The pin (14) extends through the driving ring (30) and the column 19 (100) as well as the stop inside the column (100) and is moved along a slot (13) 20 defined in the column (100). A rotating sleeve (50A) is rotatably sandwiched 21 between the driving ring (30) and the supporting seat (40). The rotating sleeve 22 (50A) has a first set of teeth (51A) formed on an outside bottom portion of the 23 rotating sleeve (50A), a second set of teeth (52A) and a third set of teeth (53A) to correspond to the two extensions (31) of the driving ring (30). Both the second 24

1 set of teeth (52A) and the third set of teeth (53A) are formed on an outer 2 periphery of the rotating sleeve (50A) and inclined relative to the first set of teeth 3 (51A) and opposite to one another. That is, the first set of teeth (51A) is linearly formed on the outer periphery of the rotating sleeve (50A) and horizontal to the 4 5 radius of the rotating sleeve (50A). However, the second and third sets of teeth 6 (52A,53A) are formed in a descending manner such that both the second set of 7 teeth (52A) and the third set of teeth (53A) are inclined relative to the first set of ٠8 teeth (51A). e gate et a production 19 With reference to Figs. 4 and 5, the supporting seat (40) has a wedge (43) 10 formed on an inner periphery of the supporting seat (40) to correspond to a tooth of the first set of teeth (51A). A recovery spring (36) is provided between the 11 1.2 supporting seat (40) and the rotating sleeve (50A). Furthermore, a handle (42) is 13 pivotally connected to the supporting seat (40) and the pin (14) extends through both sides of the handle (42). Thus pivotal movement of the handle (42) is able to 14 15 drive the driving ring (30) to move up and down relative to the column (100). 16 It is to be noted that when the two extensions (31) are received in 17. corresponding teeth in the second and the third sets of teeth (52A,53A) 18 respectively, the wedge (43) corresponds to a slope of the first set of teeth (51A). 19 When the two extensions (31) are away from the corresponding teeth of the 20 respective second and third sets of teeth (52A,53A), the release of the recoil 21 force from the recovery spring (36) forces the rotating sleeve (50A) to move 22 upward, which allows the wedge (43) to be received in a corresponding tooth of 23 the first set of teeth (51A). Before the wedge (43) is received in a corresponding

3

tooth of the first set of teeth (51A), the wedge (43) aligns with a slope of the

24

corresponding tooth of the first set of teeth (51A). Therefore, after the wedge (43) 1 is received in the corresponding tooth of the first set of teeth (51A) due to the 2 upward movement of the handle (42) to drive the driving ring (30) to move 3 upward accordingly, the rotation of the rotating sleeve (50A) makes a slope of 4 the respective tooth of the second set of teeth (52A) and the third set of teeth 5 (53A) to align with the extensions (31). When the handle (42) and the driving 6 ring (30) are lowered, the rotating sleeve (50A) is also lowered due to the 7 abutment of the two extensions (31) to bottom faces of the corresponding teeth es . 8. 9. of the second and third sets of teeth (52A,53A): Sliding movement of the two 10 extensions (31) on the slopes of the corresponding teeth of the second set of teeth (52A) and the third set of teeth (53A) causes the wedge (43) to misalign with a 11 tooth of the first set of teeth (51A). After the two extensions (31) are received in 12 the corresponding teeth of the second and third sets of teeth (52A,53A), the 13 interaction between the driving ring (30) and the rotating sleeve (50A) resumes 14 15 the original relationship as previously described. 16: Therefore, the up-and-down movement of the rotating ring (30) due to the pivotal movement of the handle (42) is able to drive the rotating sleeve (50A) 18 to rotate clockwise or counterclockwise depending on which of the slopes of the first set of teeth (51A) the wedge (43) aligns with. Further, because the second 19 set of teeth (52A) and the third set of teeth (53A) are inclined relative to the first 20 21 set of teeth (51A), the rotation of the rotating sleeve (50A) allows the two extensions (31) to be received in different teeth of the second set of teeth (52A) 22 and the third set of teeth (53A) respectively. Therefore, the downward force of 23 the rotating sleeve (50A) to the recovery spring (36) is variable, which also 24

1 means that the position of the driving ring (30) is changeable and thus the

2 extension of the spring inside the column (100) and around the transmission rod

(11) is also changeable. The purpose of changing the recovery force to the paddle

4 (20) is thus accomplished.

3

5

6

7

9

10

11

12

13

14

15

16

17

With the recovery force adjusting device of the present invention, the user is able to adjust the recovery force of the spring to the paddle by pivoting the handle (42) to achieve the purpose of changing the position of the driving ring (30). Thus after the position of the driving ring (30) is changed, the compression ,8., to the spring is changed and the force required to pivot the paddle (20) is also changed.

It is to be understood, however, that even though numerous characteristics and advantages of the present invention have been set forth in the foregoing description, together with details of the structure and function of the invention, the disclosure is illustrative only, and changes may be made in detail, especially in matters of shape, size, and arrangement of parts within the principles of the invention to the full extent indicated by the broad general meaning of the terms in which the appended claims are expressed.

1